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PRESSURE DROP STUDY OF POLYMERIC HOLLOW FIBER HEAT EXCHANGERS

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Heat Transfer and Fluid Flow Laboratory

Heat Transfer and Fluid Flow Laboratory is a part of Faculty of Mechanical Engineering, Brno University of Technology.

Heatlab Interest:

- Secondary Cooling in Continuous Casting
- Roll Cooling
- Descaling
- Product Cooling (Heat Treatment)
- Testing of Automotive Headlamps
- **Hollow fibers**

Polymeric Hollow Fiber Heat Exchangers

- Firstly published in 2004 – shell'n'tube type
- Heat transfer surface consists of hundreds of capillaries
- Small diameters, $d_o = 0.4-1.6 \text{ mm}$
- Big heat transfer surface in small volume
- Need of separation → chaotic bundles
 - Straight 214 W, chaotic 585 W – with same heat transfer area, temperature and flow conditions (Raudensky et al. 2017)
- Application: desalination, heat recuperation



Raudenský, M., Astrouski, I. a Dohnal, M. Intensification of heat transfer of polymeric hollow fiber heat exchangers by chaotisation. *Applied Thermal Engineering*. 2017, roč. 113. S. 632–638. ISSN 13594311.

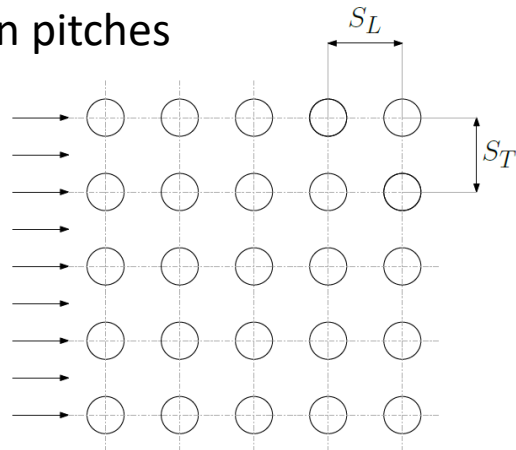
PHFHE – gas-liquid type

- Presented in 2016
- Rectangular shape
- Separation due to the textile interweaving
- Comparison with automotive radiator (Krasny et al.)
 - Heat transfer rate up to 10 kW (10 m/s, $\Delta T = 40^\circ\text{C}$),
 $U_o = 335 \text{ W}/(\text{m}^2 \text{ K})$
 - OHTC similar to the aluminum radiator
 - Higher pressure drop

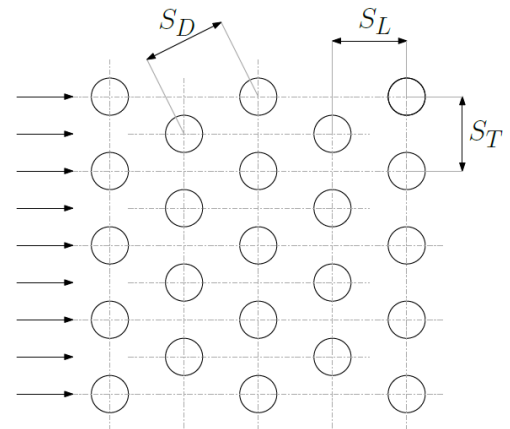


Air-side pressure drop

- Common geometry - bank of tubes
- Pressure drop is dependent on the geometry and working fluid
- Empirical models – Grimison, Jakob, Kays and London, Zhukauskas
- Two basic alignment – inline and staggered
- Most of models only for common pitches



In-line



Staggered

Pressure drop of PHFHE

Can we use the models to compute pressure drop of polymeric hollow fiber heat exchangers?

- There is at least one – published in (VDI-Heat Atlas) based on Gnielinski
- Pressure drop given by the equation:

$$\Delta p = \xi n_{mr} \frac{\rho v_{max}^2}{2}$$

- where pressure drop coefficient is function of Reynolds number, pitches and tube diameter

$$\xi = f(\text{Re}, a, b, c)$$

The experimental details

Experimental data from (Krasny et al. 2016) compared to the VDI-HA model

The experimental details

- Two PHFHE with different diameters
- Inside fibers 50/50% water-glycol coolant solution, temperature 60°C
- Cooling medium: Air, 20°C

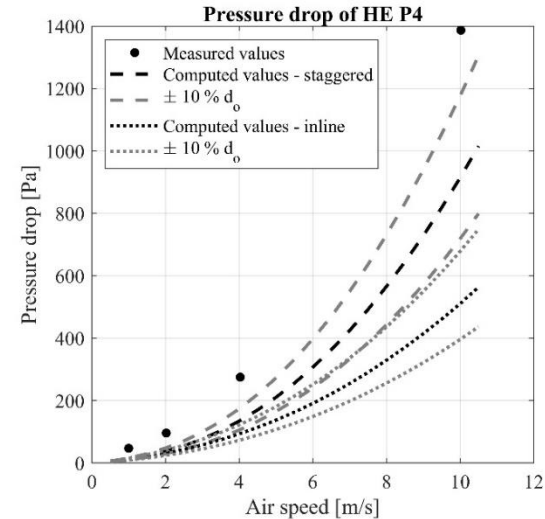
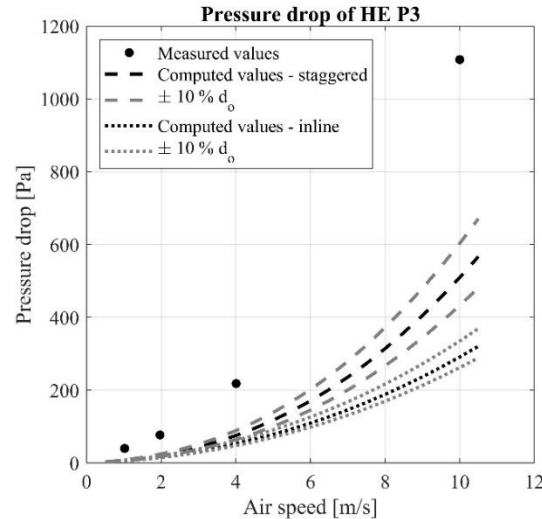
Due to the extruding technology the diameter of fiber varies $\pm 10\%$

PHFHE	l	d_o	d_i	S_T	S_L	n
	[mm]	[mm]	[mm]	[mm]	[mm]	[-]
P3	250	0,6	0,48	1,8	2	14
P4	220	0,8	0,64	1,8	2	14

Tested PHFHEs details

Results

- Computed values underestimated
- Relative error up to 458 % for P3 and 280 % for P4
- Significant difference due to the diameter variation (up to 32 %)
- Why?
 - Flexibility of fibers – the pitches are changing during operation of heat exchanger
 - Flow-induced vibrations



Conclusion

- PHFHE are alternative to conventional heat exchanger which is chemically resistant.
- They have big heat transfer surface to volume ratio.
- There is not much research about gas-side pressure drop.
- This study shows that discrepancy between the model and experiment is huge.
- There is need to revise/set new correlations for flexible tubes.

**THANK YOU FOR YOUR
ATTENTION**





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BRNO UNIVERSITY OF TECHNOLOGY
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Project reg. Nr. CZ.02.1.01/0.0/0.0/16_019/0000753



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